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AN EVALUATION OF GREASE-TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS (Final Status Report No. 8)

By E. L. McMurtrey Materials and Processes Laboratory

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George C. Marshall Space Flight Center

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operate for long periods of lubricants, a series of tests in R-4 size bearings in five tions of each test were made used to select four lubricant repetitions of each test for sets have been completed. (2) start-stop operation, wit tinuous vacuum operation at	spacecraft or space stations will time in environments which are has been completed to evaluate different environments for a 1-to provide statistical samples. In this comes for 5-year tests in selected estatistical samples. In this comes the three 5-year tests in (1) of the both in vacuum at ambient te 93.3°C have been completed. In all environments have been lipolyether (PFPE) grease.	adverse to most 38 grease-typ year period. These tests with the program on tinuous oper mperatures, as In both the 1-	st bearing be lubricants Four repeti- were also ith five 1, 172 test ration and nd (3) con-
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TECHNICAL MEMORANDUM

AN EVALUATION OF GREASE TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS (Final Status Report No. 8)

I. INTRODUCTION

This is the eighth and final report in a series of status reports to be issued covering a long-term test program to evaluate a number of fluid lubricants in ball bearings operating under various environmental conditions. The first report [1] discussed the general test program and gave the results of the first series of vacuum ambient temperature tests. Since that report, sufficient progress has been made to provide a comparison of many of the greases evaluated for ball-bearing lubricants in different environments; therefore, it is believed that the information also contained in the subsequent reports [2,3,4,5,6,7] will prove useful to those responsible for selecting lubricants for various space missions.

This program is an extension and expansion of pioneering work done by Young et al. [8] on fluid lubricated bearings operating in vacuum. Because many of the spacecraft planned for the future will require mechanisms that can operate for long periods of time in adverse environments, it was necessary to define the operating limits of available lubricants in these environments. As of October 1984, 680 sets of 1360 bearings have completed 1 year of testing and 60 sets of 120 bearings have completed 5 years of testing. The plan was to continue the test program using commercially available greases to determine statistically which lubricants would provide maximum bearing operating life with the environmental conditions under which they may be used. This procedure was used to eliminate all but four candidate lubricants for 5-year tests. These lubricants have been tested under selected environmental conditions to failure or for the 5-year period.

II. TEST EQUIPMENT

To provide a statistical sample of a number of lubricants operating under various environmental conditions, it was necessary to conduct a large number of tests simultaneously. Therefore, 20 test motors, each containing two test bearings, were set up in each chamber. Each test set consisted of four samples (eight bearings) of five different lubricants for the 1-year tests. One test set is shown in Figure 1. The bearings chosen for testing were size R-4, 0.635 cm I.D. by 1.59 cm O.D. (0.25 in. I.D. by 0.625 in. O.D.), 440 C steel (RC 60-65) with ribbon type stainless steel cages. An approximate 25 to 30 percent fill of the candidate greases was applied to each bearing, unless otherwise specified.

The motors used in these tests have the following characteristics:

- 1) Type ac hysteresis, single phase, 60 cycle
- 2) Speed 3600 rpm, synchronous
- 3) Current -0.22 Amp.

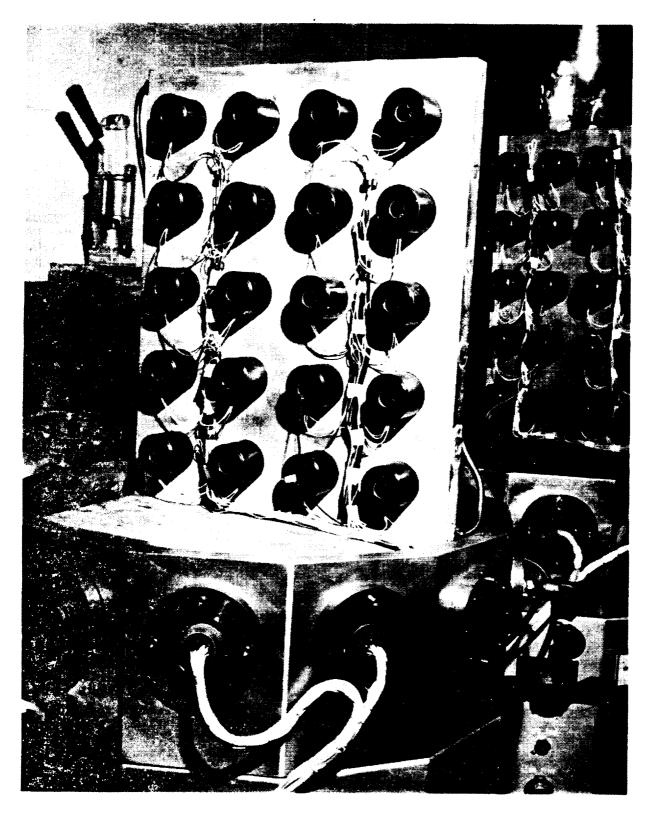


Figure 1. Test motors in vacuum chamber with bell jar removed.

Because these motors do not use brushes, no problems were encountered with brush dust contamination of the bearings. In addition, these motors use approximately the same current when stalled as when operating at 3600 rpm; consequently, a bearing failure does not cause motor damage from overheating. A disassembled motor bearing set is shown in Figure 2.

To control temperature, the motors are mounted in an aluminum plate which is furnished with passages so that thermal control fluids (water or liquid nitrogen) may be used to control the motor temperature. Temperature is measured by thermocouples attached to the mounting plate and to selected motor cases.

Each mounting plate with its motor set is placed in a glass bell jar vacuum system. These bell jars are part of a 12-position vacuum system which is capable of maintaining pressures in the $1.3 \times 10^{-4} \ \text{N/m}^2$ (1 × 10^{-6} torr) range during test operation. The same bell jars are used for the oxidation and low temperature start tests.

III. TEST PROCEDURE

Since most bearings operating in space are not subject to a radial load, the major load to the test bearings is a thrust load applied by a wave washer. The motors, specially ordered from the manufacturer, are shimmed to maintain a 2.27 kg (5 lb) thrust load on both bearings. This is equivalent to a 1.28×10^9 N/cm² (185 000 psi) Hz stress on the balls and inner races. The 3600 rpm speed allows 216 000 rev/h on each bearing until failure. Each bearing which survives the 1-year test will have completed approximately 1 892 000 000 revolutions.

At the beginning of the test program, 25 lubricants from seven general chemical classes were selected for evaluation, with 13 lubricants being added after the test program had begun. These lubricants were selected to represent most of the military grease specifications, as well as special nonspecification materials which had shown promise in space applications. The code designations given do not necessarily indicate different chemical compositions; the greases designated PFPE-4, PFPE-5, and PFPE-6 are from the same supplier, but with different base oil viscosities.

A general description of these greases is given in Table 1. It was planned to add additional lubricants to the test program (13 lubricants have been added since the start of the program) if data on new lubricants indicated that they had characteristics that would make them good candidates for one or more of the environments used in the test program.

The environments for the test program were as follows:

- 1) $10.134 \times 10^4 \; \text{N/m}^2$ (14.7 psi) air at 90 percent relative humidity (oxidation tests)
- $^{2)}$ 6.894 \times 10 4 N/m 2 (10 psi) $\rm O_{2}$ at 90 percent relative humidity (oxidation tests)
 - 3) Vacuum, ambient temperature (38°C)
 - 4) Vacuum, high temperature (93.3°C)
 - 5) Vacuum, ambient temperature, with start-stop operation
 - 6) Low temperature start.

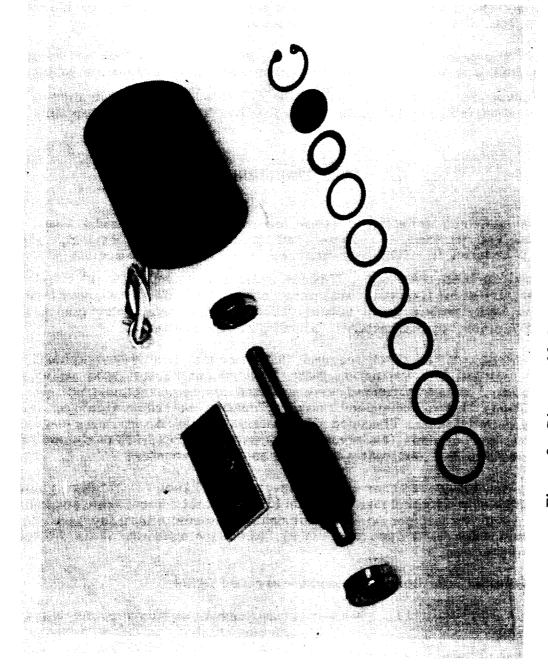


Figure 2. Disassembled ac motor with R-4 bearings.

TABLE 1. DESCRIPTION OF TEST LUBRICANTS

	1					_					_																
Description of Greases	Instrument Brg. Brg.	Hi Temp Acft General Purpose	General Furpose General Purpose	Oscillating Brg. Brg., Wide Temp, Range	Vacuum	Brg., Wide Temp. Range Hi Temp. Acft	Hi Temp. Corr. Resistant	Brg., Vacuum	Brg., Wide Temp.	Wide Temp. with MoS2	Acft. Instrument	Acit. Instrument Hi Temp Acft	Low Temp.	Brg., Vacuum	Vacuum Hi Toma Boll nam	Ball and Roller Brg.	General Purpose	Acit, and Instrument	had, nest, Brg, Experimental Vac. Low Speed Brg,	Chem. Inert Brg.	Chem. Inert Hi Temp.	H1 Vac. Brg. Chem. Inert Hi and Low Temp	data more and the control of the con	Chem. Inert Brg.	Low Temp.	Chem. Inert Vacuum, Hi Temp. Chem. Inert Vacuum, Hi Temp.	
Oil Viscosity Index	101 110							107				160	1	137				-			001	350		110	23	134	-
38C Oil Viscosity (cs)	158 400 300	000	•	38	tu tu	cc	108	119.7	9	14	14	162	11.8	27.5	000						404	129	_	153	1 × × × × × × × × × × × × × × × × × × ×	270	
Thickener	Inorganic Inorganic Microgal	Li Soap MoSa-Nonsoan	MoS2-Nonsoap	Microgel	None	Nonsoap	Synthetic	Orapnite-Lead Na Soap	Arylurea	Microgel	Microgel	Li Soap	Li Soap	Graphite-Lead	Li Soap	Organic Dye		Silica	Silica		Fluorotelomer	Fluorotelomer	Fluorotelomer	Fluorotelomer	Fluorotelomer	Fluorotelomer	Fluorotelomer
Gen. Chem. Class of Base Oil	Highly Refined Mineral Highly Refined Mineral Mineral	Mineral Mineral	Mineral Mineral	Synthetic Mineral	Straight Chain Hydrocarbon Mineral-Diester	Synthetic Hydrocarbon	Mineral Mineral	Mineral	Diester	Diester	Diester		Synthetic Ester	Silicone	Silicone	Silicone	Silicone	Silicone	Fluoro-Silicone	Fluoro-Carbon	PFPE	PFPE	PFPE	7.7.7.0 1.0.0.0 1.0.0.0	PFPE	PFPE BEDE	FFE
MIL Spec	83176 3545B	10924B 23549A	23549B 25537A	25760A		81322			25760A	23827A		1	23827A			25013D	_			•							
Lubricant Code	M-1 M-2 M-3	M-4 M-5	M~6	M-7	8 - S W = S	M-10	M-11	M-13	ES-1	ES-3	ES-4	ES-5	ES-6	Si-1	Si-2	Si-3	Si-5	Si-X	FS-1 FS-2	FCC-1	PFPE-1	PFPE-2	DEDE- 2	PFPE-4	PFPE-5	PFPE-6 PFPE-7	1-711
Manufacturer Designation	KG80 SRG 200 Aeroshell 5	Royco 24R Royco 49	Royco 49B Aeroshell 14	Aeroshell 16	Unitemp 500	Mobilgrease 28	BP 2110	Exxon Andok C	Aeroshell 17	Aeroshell 7	L-11G	Exxon 5182	BP 8135	DC No. 33		Supermii 31052 G-330M	G-341L	3L27-2	FS-1281	Kel-F No. 90	803	3L-38RP Rebed*		240AZ	240AB	240AC 3L-38-MS	

* Vacuum baked at 100°C (212°F) for 20 hr.

The fine: status of the test program is given in Table 2.

The evaluations for all tests, except the low temperature tests, were based primarily on a go/no-go system. The motor torque is low and the inertia of the system is low; therefore, when the bearing tends to seize, the motor stops without further damage to the bearings. The following data are taken during the test:

- 1) Total test time
- 2) Vacuum or atmospheric conditions
- 3) Temperature
- 4) Total cycles, if appropriate.

The bearings are weighed before and after testing, and the percent of weight loss of lubricant is calculated.

In the low temperature start tests, the motors are installed in the cooling plate, and the system is evacuated to prevent frost formation. LN $_2$ is circulated through the cooling plate. The temperature is measured with thermocouples in contact with the outer race of the front bearing. Before cooling is initiated, the motors are operated for 30 min to channel the grease. The temperature is then dropped to $^{-100^{\circ}\text{C}}$ and held approximately 30 min. The temperature is then allowed to rise slowly using a thermocouple on the mounting plate for control. After each 3°C rise, the motors are switched on for approximately 5 sec, and the temperatures of the front bearings are recorded. When each motor starts and comes up to full speed, the front bearing temperature is used as the low temperature starting capability of the lubricant. The starting torque of the motors used in this test is 1.05×10^{-2} N m (1.5 in. oz). Each low temperature test is repeated at least twice, and an average temperature is taken of the four motors and two tests.

IV. TEST RESULTS

A. Low Temperature Start Tests

Twenty-six of the candidate lubricants have been evaluated for low temperature start capability. Unfortunately, the temperature at which the bearings will stall is a function of the volume of grease in the bearing, as well as the viscosity of the grease; therefore, some variation in stall temperature is sure to occur. To help overcome this difficulty, four motors are tested with each lubricant and at least two tests are made on each motor. The resulting stall temperatures are then averaged. Results of these tests are shown in Table 3. Ordinarily, the vacuum stability requirements and the low temperature starting torque requirements are mutually exclusive because a low viscosity fluid provides better low temperature capabilities and a high viscosity fluid tends to be more vacuum stable. The results of these tests are; therefore, rather surprising since the PFPE-2 grease, which has a 38°C viscosity of 130 cs, has superior low temperature capabilities and is also one of the most vacuum stable greases evaluated. These capabilities are somewhat more understandable when it is noted that the base oil for this grease has a viscosity index of 350 and a molecular weight of over 9000.

TABLE 2. FINAL STATUS OF LUBRICANT TESTS

					Test Cond	litions	
			dizing onment d	Vacuum (38°C)	Vacuum (93.3°C)	Vacuum (Start- Stop)	Low Temperature Start
KG 80	M-1	a	a	a	a	a	a
SRG 200	M-2			a	a	a	a
Aeroshell 5	M-3	а	a	a,b	a,a,b	a,a,b	a
Royco 24R	M-4			a			a
Royco 49	M-5			a	а	а	a
Royco 49B		a	a			a,a	
Aeroshell 14	M-6	İ		a	İ		a
Aeroshell 16	M - 7			a			
Apiezon L	M-8			a			
Unitemp 500	M-9			a			
Mobilgrease 28	M-10	a	a	a	a	а	
Conoco HD #2	M-11	a	а	а	a	a	a
BP 2110	M-12	a	a	a	a	а	a
Andok C	M-13	a	a	а	a	а	a
Supermil 06752	ES-1	a		a		а	a
Aeroshell 17	ES-2			a			
Aeroshell 7	ES-3				а	a	a
L-11G	ES-4			a			a
Exxon 5182	ES-5	a	а	a	a	a	a
Exxon 325	ES-6	a	а	а	а		a
BP 8135	ES-7	a	а	a	a	a	а
DC No. 33	Si-1	a		а			
G-351	Si-2	а	а	a,b	a,a,b	a,a,b	а
Supermil 31052	Si-3	1		a	a	a	a
G = 330M	Si-4			а	a		а
G-341L	Si-5	а	а	а	а	8.	а
3L 27-2	Si-X	a		а	а		
FS-1281	FS-1	а		a			
FS-1290	FS-2	а	а	а	a,a	а	
Kel-F No. 90	FCC-1				а		
803	PFPE-1	а	а	a,b	a,a,b	a,a,b	a
3L-38RP	PFPE-2	а	а	a,b	a,a,a,b	a,a,b	a
3L-38RP Baked*		а		a	а	a	a
631A	PFPE-3	а	а	a	а	a	a
240Az	PFPE-4	а	а	a	а	а	a
240AB	PFPE-5	а	а	а	a	а	
240AC	PFPE-6	а	a	a	а	a	a
3L-38 MS	PFPE-7	а	а	а	a	a	а

<sup>a. Test complete, 1 year or 2 days (low temperature test only)
b. Test complete, 5 year
c. Air, 90% RH
d. 10 psi O₂, 90% RH</sup>

^{*}Vacuum baked at 100°C for 20 hr.

TABLE 3. LOW TEMPERATURE START, °C

Lubricant	1	2	3	4	Average
Si-3	-62.8	-78.9	-76.1	-70.0	-71.9
PFPE-7	-68.6	-68.6	-68.6	-68.6	-68.6
PFPE-2	-61.4	-57.5	-72.5	-82.2	-68.4
PFPE-2 Baked	-68.1	-66.7	-64.7	-64.7	-66.0
M-4	-58.9	-70.8	-60.0	-58.9	-62.1
M-6	-56.7	-55.0	-60.3	-60.3	-58.1
ES-4	-53.9	-57.8	-55.8	-55.0	-55.6
ES-1	-51.1	-53.8	-51.1	-51.1	-51.8
Si-5	-49.2	-49.2	-49.2	-49.2	-49.2
ES-3	-53.9	-41.1	-56.1	-42.1	-48.3
PFPE-1	-44.3	-44.3	-49.4	-48.0	-46.5
ES-5	-42.5	-42.5	-46.4	-46.4	-44.5
ES-7	-43.6	-42.8	-43.6	-43.6	-43.4
M-12	-42.8	-42.8	-42.8	-44.2	-43.2
ES-6	-41.4	-41.4	-41.4	-41.4	-41.4
PFPE-4	-36.1	-36.1	-36.1	-36.7	-36.3
Si-4	-34.4	-34.4	-34.4	-34.4	-34.4
M-13	-30.3	-31.7	-30.3	-30.3	-30.7
M-5	-23.1	-20.3	-26.4	-21.1	-22.7
M-11	-21.9	-21.9	-21.9	-21.9	-21.9
Si-2	-16.7	-16.7	-16.1	-16.1	-16.4
M-3	-16.1	-10.3	-16.1	-18.1	-15.2
M-1	- 6.7	- 4.4	- 4.4	- 4.4	- 4.98
PFPE-6	- 4.4	- 4.4	+ 1.1	- 4.4	- 3.02
PFPE-3	- 0.56	0.0	0.0	0.0	- 0.14
M-2	+ 3.30	+ 3.30	- 8.30	+ 3.30	+ 0.40

B. Continuous Vacuum Ambient Temperature Tests

Ten 1-year tests have been completed; the results are given in the first part of Table 4. Sixty-four motors (16 lubricants) have had no failures resulting from lubricant depletion, but motor No. 3 of lubricant M-3 had a drive motor failure. Also, the first 13 lubricants listed have had less than a 20 percent average weight loss.

One 5-year test has been completed; the results are given in the second part of Table 4.

The average temperatures (ten 1-year tests) have been as follows:

Front bearing - 96°F (35.6°C) Rear bearing - 143°F (61.7°C) Mounting plate - 73°F (22.8°C).

The average temperatures (one 5-year test) have been as follows:

Front bearing $-107^{\circ}F$ (41.7°C) Rear bearing $-134^{\circ}F$ (56.7°C) Mounting plate $-74^{\circ}F$ (23.3°C).

TABLE 4. RESULTS OF VACUUM TESTS AT 38°C

		Но	urs to	Failure ^t	1			Weig	t Loss	s (§) ^b		
Lubricant	1	2	3	4	Aver	age	1	2	3	4	Ave	rage
PFPE- 2	8760	8760	8760	8760	876	0	5.1	6.9	8.1	5		6.3
Si-2	8760	8760	8760	8760	876		3.5	12	6	4.5		5.5
M-5	8760	8760	8760	8760	876		7.5	5	8	6.5		3. 3 3. 8
PFPE-2 ^d	8760	8760	8760	8760	876		7.7	5.4	8.8	5.7		5. 9
Si-4	8760	8760	8760	8760	876		9.4	8.6	5.7	5.7		7.4
ES-5	8760	8760	8760	8760	876		7.6	8.6	6.4	7.5		7.5
M-12	8760	8760	8760	8760	876		6.5	12.4	12.6	6.1		9.4
PFPE-6	8760	8760	8760	8760	876		6	13.5	12.5	7). 4). 8
M-3	8760	8760	С	8760	876		5.9	13.1	11.9	8.2		
PFPE-3	8760	8760	8760	8760	876		10	15.5	8.5			9.8
FS-2	8760	8760	8760	8760	876		7	21	8.5 17.5	8 10.5		1.5
ES-7	8760	8760	8760	8760	876		14.3	13.7			14	
PFPE-1	8760	8760	8760	8760	876		10.5	33	12	16.6		. 2
M-10	8760	8760	8760	8760	876		26		15	17		. 9
M-13	8760	8760	8760	8760	8760			20.5	19	23		.1
M-2	8760	8760	8760	8760	8760		28	41.8	31.9	28.2		. 5
M-11	8513	8760	8760	8760			66	49	40	50		. 3
Si-5	4739	8760	8760	8760	8698		20.1	19.6	15.4	22	19	
PFPE-7	4397	8760	8760		7755		9.5	5.4	11.4	3.1		. 4
M-1	8760	8760	3700	8760	7669		27.2	6	2.5	2.3		. 5
Si-1	8760	8760		8760	7495		21.5	27.5	23	25	24	. 3
PFPE-4	684		1709	8760	6997		35	25	41	22.5	30	. 9
ES-1	3524	8760	8760	8760	6741		26	11.5	13	9	14	. 9
M-7	2530	8760	8437	4397	6280		24.5	39.5	23.5	18.5	26	. 5
PFPE-5		8760	8760	3367	5854		53.8	46.8	54.3	41.9	49	. 2
Si-X	2096	3517	8760	8760	5783		33.5	40.5	3.5	3.5	20	
M-8	1041	6015	8760	5710	5382		27.5	28	40	47.5	35	
ES-6	392	8760	8524	1976	4913		3.3	0.8	0.8	22.5		. 9
M-9	3563	5199	8760	1894	4854		61	67.8	59.6	68.3	64	
	2543	1487	1199	8760	3497		34.2	27.6	49.3	24.4	33.	
Si-3	5613	2164	1659	456	2473		52.5	27	43.5	24.5	36	
M-4	2671	859	311	160	1000		74.5	73.5	82	78	77	
ES-2	427	696	743	9 11	694		61.4	56.1	72.3	61.8	62.	Q.
ES-4	559	593	559	823	634		30.5	32.5	39	41	35.	
FS-1	174	245	831	511	440	ļ	7.5	14.5	22.5	15.5	35. 15	U
M-6	473	219	336	286	329	- 1	67	76	68.5	70.5	70.	5
		H	ours to	Failur	e ^a				eight L			
						Aver-						
Lubricant	1	2	3	4	5	age	1	2	3	4	5	Aver- age
PFPE-1	319 18	22676	43800	21140	32173	30341	50 1	20 =		40.0		
PFPE-2	43800	43800	43800	43800	43800		52.1		7.51	43.2	46.7	36.4
VI - 3	43800	43800	43800	43800		43800	7.2		8.5	12	9.9	10.7
Si-2	19323	21424	32086	43800	43800	43800	15.9		11.4	9.6	13.7	13.8
	10020	71374	32000	42000	1411	23609	35.7	33.5	47.7	10.4	37.6	33

<sup>a. Or to end of test (1 year = 8760 hr and 5 years = 43800 hr).
b. Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4 or motor Nos. 1 through 5).</sup>

c. Drive motor failed.

d. Baked in vacuum at 100°C for 20 hr.

C. Continuous Vacuum High Temperature Tests

Seven 1-year tests have been completed; the results are given in the first part of Table 5. Forty-four motors (11 lubricants) have had no failures resulting from lubricant depletion, but motor No. 2 of lubricant M-2 had a drive motor failure. Also, the first seven lubricants listed have had less than a 20 percent average weight loss.

One 5-year test has been completed; the results are given in the second part of Table 5.

The temperature in these high temperature tests is controlled by regulating the cooling water supply to the mounting plate so as to maintain its temperature at 65.5°C (150°F). The average temperatures (seven tests) have been as follows:

```
Front bearing - 170°F (76.7°C)
Rear bearing - 203°F (95.0°C)
Mounting plate - 153°F (67.2°C).
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The average temperatures (one 5-year test) have been as follows:

```
Front bearing -175^{\circ}F (79.4°C)
Rear bearing -189^{\circ}F (87.2°C)
Mounting plate -150^{\circ}F (65.5°C).
```

D. Continuous Oxidation Ambient Temperature Tests

During the development of the Skylab thermal control fan, problems were encountered with bearings operating in a highly oxidizing atmosphere; therefore, it was believed that a highly oxidative environment should form a part of the present evaluations.

The first set of tests was made in air at 90 percent relative humidity. However, it appeared that a pure oxygen environment might be more severe; therefore, an additional set of tests was made in 10 psi pure oxygen at 90 percent relative humidity. Although no temperature measurements were made during these two tests, the bearing operating temperatures have been relatively close to subsequent ambient temperature tests, since the operating procedure for controlling cooling water flow to the motor mounting plates has been identical. Thermocouples were added on the subsequent seven tests.

Nine 1-year tests have been completed; the results are given in Table 6. Forty-four motors (eleven lubricants) in the air tests have had no failures resulting from lubricant depletion, but motor No. 3 of lubricant Si-1 had a drive motor failure. Also, the first eight lubricants listed in these air tests have had less than a 20 percent average weight loss. Sixty motors (fifteen lubricants) in the oxygen tests have had no failures resulting from lubricant depletion, but motor No. 3 of lubricant PFPE-3 had a drive motor failure. Also, the first six lubricants listed in these oxygen tests have had less than a 10 percent average weight loss.

The average weight loss of the 25 air tests is 19.9 percent. The average weight loss of the 20 oxygen tests is 18.6 percent. The air tests are more severe than the oxygen tests. Since this trend continued, the original assumption that oxygen tests might be more severe was incorrect.

TABLE 5. RESULTS OF VACUUM TESTS AT 93.3°C

	ļ	- I	lours to	Failur	e ^a			Weigh	t Loss	(%) b	
Lubricant	1	2	3	4	Average	,	1	2	3	4	Average
PFPE-2 PFPE-2 ^d	8760	8760	8760	8760	8760	1	3	13.5	14	17	14.5
	8760	8760	8760	8760	8760	1	4.2	14.2	17.3	14.4	
PFPE-6	8760	8760	8760	8760	8760	1	9.5	9	19.5	13.5	
PFPE-5	8760	8760	8760	8760	8760	1	4	21.5	12	15.5	
PFPE-1	8760	8760	8760	8760	8760	1	8	12.5	24.5	12	17
M- 5	8760	8760	8760	8760	8760	1	. 5	24.5	14.5	15.5	
PFPE-3	8760	8760	8760	8760	8760	1	8	16.5	24	19	19.5
$^{11-36}$	8760	8760	8760	8760	8760		7.4	25	24.6	23	25
M-3	8760	8760	8760	8760	8760		9.5	35	27	34.5	
M-1	8760	8760	8760	8760	8760		9	37	32	43	35.5
M-2	8760	c	8760	8760	8760		5	31	50	47.5	
FS-2	6813	8760	8760	8760	8273		9	35.5	30.5	35	40.5
M-5e	4979	8760	8760	8760	7815		1.6	28.3	15.4	11.4	
M-12	8760	4745	8760	8760	7756		8	42.6	29.3	34.5	
PFPE-2e	4979	8760	6659	8760	7290		9.3	12.9	40	4.9	
Si-2	8760	2870	8760	8760	7288		3	51	23.5	36	33.4
PFPE-1e	6980	8760	8760	4187	7172		7.6	10.7	11.4	28.5	
M-11	8760	5658	2432	8760	6403		4.9	41.7	23.7	43.6	
Si-4	1218	8760	7940	6609	6132		0.5	9	27	25	
Si-2e	4691	8760	8760	2156	6092		0.5	20.4	17.9	19.9	27.9
PFPE-7	2073	2057	8760	8760	5413	5		49.5	26.3		
Si-5	8760	755	515	8760	4698		6.7	11.8		16.3	
M-13	1905	1673	1362	5995	2734		0.7		12.2	10.7	
ES-5	2432	1445	4442	1327	2412			67	60.2	75.1	
Si-3	686	2290	1702	2327	1751		3.8 7.5	32.7	40.8	34.9	
PFPE-4	3193	350	2523	282	1587			41	48.5	35.5	
M-10	1091	1338	$\begin{array}{c} 2323 \\ 2222 \end{array}$	$\begin{array}{c} 282 \\ 1274 \end{array}$		5		39	63	44	50
ES-6	1031	1761	729	594	1481		8.7	73.8	48.3	63.3	63.5
FCC-1	353	1280	521		1029		3.9	73.6	79.1	61.9	74.6
Si-X	174	101	321 1047	166	580	4		53	47.5	54	50.5
ES-7	161	57	1047	68.5			0.5	59.5	56	62.5	62.5
ES-3	82	73		177	130		4.7	56	56.9	85.5	63.3
E0. 0	02		70	71	74	8	5.5	91.5	83.5	88	87.1
			Hours	to Failu	re ^a			Weigh	t Loss	(%) b	
Lubricant	1	2	3	4	5 Ave	rage	1	2	3	4	5 Average
PFPE-1	27063	3971	5754	9012	26278 14	4416	26.3	28.7	22.8	23.7	35.8 27.5
PFPE-2	С	38749	26647	43800		7912	14.4		53.6	15.7	42.6 41.5
M-3	38764	37886	26285	19557		3258	40.2	49.6	33.2	51.8	24.5 39.9
Si-2	17877	25881	1759	21393		7437	35	53.3	38.8	44.2	42.3 42.7

a. Or to end of test (1 year = 8760 hr and 5 years = 43800 hr).

b. Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4 or motor Nos. 1 through 5).

c. Drive motor failed.

d. Baked in vacuum at 100°C for 20 hr.

e. 10-15 percent fill, all others 25-30 percent fill.

TABLE 6. RESULTS OF OXIDIZING TESTS

	γ		14.7 p	si Air a	at 90% Relat	ive Hu				
		Но	urs to	Failure	a —————		We	ight Lo	ss (%))
Lubricant	1	2	3	4	Average	1	2	3	4	Averag
PFPE-1	8760	8760	8760	8760	8760	5	5.5	5	5.5	5.3
PFPE-3	8760	8760	8760	8760	8760	5.9	5.4	7.2	6.3	6.2
M-3	8760	8760	8760	8760	8760	6.8	5.7	6.3	9.6	7.1
FS-2	8760	8760	8760	8760	8760	4.8	8.5	9	8.5	7.7
Si-5	8760	8760	8760	8760	8760	8	7.1	12.2	6.6	8.5
ES-1	8760	8760	8760	8760	8760	12.5	12	11.5	12	12
M-10	8760	8760	8760	8760	8760	11.9	12.1	9.5	16.7	12.6
M-13	8760	8760	8760	8760	8760	31.9	15.5	15.2	12.3	18.7
M-11	8760	8760	8760	8760	8760	29.9	35	26.7	38.5	32.5
Si-X	8760	8760	8760	8760	8760	35.5	40.5	43	42	40
Si-1	8760	8760	c	8760	8760	48.5	47	40	46	45.4
M-12	8688	8760	8760	8760	8742	24.3	8.9	5.8	3.9	10.7
PFPE-4	8760	8760	8760	8357	8659	25	33.5	33.6	41.4	33.3
PFPE-5	8760	7147	8760	8598	8316	8.3	35.6	5.9	30.2	20
ES-6	8760	8760	8760	6456	8184	20.8	28	32.7	47.4	32.2
M-5e	4884	8760	8760	8760	7791	32	5.7	5.2	5.9	12.2
Si-2	8760	8760	6065	5287	7218	7.6	8.5	44.1	34.7	23.7
ES-7	8760	8760	2445	8760	7181	6.9	6.6	15.2	9	9.4
M-1	8760	8760	2116	8760	7099	16.8	19.5	24.5	12.1	18.2
ES-5	1714	8760	8760	8760	6999	19.5	12.7	14.8	19.5	16.6
FS-1	8760	405	8760	8760	6671	3	3.5	3	4.5	3.5
PFPE-2	8760	7709	5699	2480	6162	19.2	36.4	34	32.3	30.5
PFPE-7	4117	8760	5699	5467	6011	47.2	10.8	30.7	49.8	34.6
PFPE-7	7938	4473	8262	3077	5938	26.1	44.5	17.3	39.7	31.9
PFPE-2d	1955	851	995	8760	3140	30.4	29.6	30.9	3	23.5
	<u>. </u>		10 ps	i Oxvge	en at 90% Re	·				
		Hot		Failure ^a		<u> </u>			ss (%)b	
Lubricant	1	2	3	4	Average	1	2	3	4	Averag
DC F	0700	0700	0880	0500		<u> </u>				
ES-7	8760	8760	8760	8760	8760	2.6	1.5	1.6	1.8	1.9
Si-2	8760	8760	8760	8760	8760	9.6	1.7	4	3.7	4.8
M-3	8760	8760	8760	8760	8760	6.3	4	6.3	6	5.7
M-5e	8760	8760	8760	8760	8760	10.3	4	3.5	7.4	6.3
M-12	8760	8760	8760	8760	8760	12.9	7.2	7	3.8	7.7
ES-5	8760	8760	8760	8760	8760	7	10.5	10.6	5.6	8.4
PFPE-1	8760	8760	8760	8760	8760	6.7	3.8	20.8	8.8	10
PFPE-6	8760	8760	8760	8760	8760	15.7	10.2	9.3	14.3	12.4
Si-5	8760	8760	8760	8760	8760	14.7	11.8	13	17.3	14.2
FS-2	8760	8760	8760	8760	8760	14.4	13.7	13.2	20	15.3
PFPE-3	8760	8760	C	8760	8760	11.3	11.6	32.4	12.1	16.9
M-1	8760	8760	8760	8760	8760	20	19	17.8	22.6	19.9
ES-6	8760	8760	8760	8760	8760	24.7	27.8	13.7	17.4	20.9
M-10	8760	8760	8760	8760	8760	19.4	22.8	21.8	20.4	21.1
PFPE-4	8760	8760	8760	8760	8760	50.9	30	70.7	39	47.7
M-13	8760	8760	8760	8369	8662	52	44.5	51	64.5	53
PFPE-7	8760	8760	8760	7946	8557	3.4	2.3	2.4	1.8	2.5
PFPE-2	8760	4795	8760	8760	7769	6.7	47.1	11.8	11.3	19.2
M-11	8369	8689	3806	8477	7335	54.1	47.8	59.6	51.8	53.3
DEDE 6	2110	6200	0760	9760	6760	20 1	21 0	04.0	0.7. 7	00.0

6399

8760

6760

8760

38.1

31.9

24.8

27.7

30.6

3119

PFPE-5

<sup>a. Or to end of test (1 year = 8760 hr).
b. Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4).
c. Drive motor failed.
d. Baked in vacuum at 100°C for 20 hr.</sup>

The average temperatures (seven 1-year tests) have been as follows:

```
Front bearing - 84°F (28.9°C)
Rear bearing - 115°F (46.1°C)
Mounting plate - 82°F (27.8°C).
```

E. Start-Stop Vacuum Ambient Temperature Tests

Since many mechanisms do not operate continuously, it was decided to simulate the boundary conditions which exist between the balls and races of a bearing during acceleration and deceleration. Timers are used to shut off the motors for 10 sec every 150 sec (24 cy/hr) or for 20 sec every 180 sec (20 cy/hr).

Seven 1-year tests have been completed/ the results for the first five are given in the first part of Table 7. Fifty-two motors (thirteen lubricants) have had no failures resulting from lubricant depletion. Also, the first nine lubricants listed have had less than a 20 percent average weight loss.

One 5-year test including the results of the last two 1-year tests for motors specified by note e has been completed; the results are given in the second part of Table 7.

Cycle counters are used at the start-stop stations to record the total number of cycles. The total cycles of the five 1-year and one 5-year tests were as follows:

- 1) 202 382
- 2) 188 342
- 3) 174 206
- 4) 177 337
- 5) 210 382
- 6) 1 051 568.

The average temperatures (seven 1-year tests) have been as follows:

```
Front bearing - 94°F (34.4°C)
Rear bearing - 119°F (48.3°C)
Mounting plate - 70°F (21.1°C).
```

The average temperatures (one 5-year test) were as follows:

```
Front bearing - 123°F (50.6°C)
Rear bearing - 170°F (76.7°C)
Mounting plate - 105°F (40.6°C).
```

V. FUTURE PLANS

Since all but four lubricants were eliminated for the 5-year test program, a rating sheet (Table 8) was devised to eliminate those lubricants which performed poorly under the various test environments. The ratings are made by assigning the number 1 to the lubricant which performs the best in a particular test, the number 2 to the second best, etc. Where several lubricants are considered equal, the positions are averaged and assigned to all the equivalent lubricants. Table 8 is used

RESULTS OF START-STOP TESTS TABLE 7.

		H	Hours to	. Failure ^a	ຜູ			We	Weight Loss	q(%) ss			Cycle
Lubricant	1	2	က	4	Αı	Average	1	2	3	4		Average	Time (s)
PFPE-6	8760	8760	8760		_	8760	00	3	6.5			5.4	180
PFPE-1	8760	8760	8760			8760	4.5	2	3.5			5.8	150
$PFPE-2^{C}$	8760	8760	8760			8760	5.7	6.1			_	6.2	180
PFPE-2	8760	8760	8760			8760	7	8.5		7.	5	7	150
ES-5	8760	8760	8760			8760	7.1	7.4	8	5.	2	7.1	180
	8760	8760	8760			8760	12	9	8.5	10.	5	9.3	180
PFPE-7	8760	8760	8760			8760	17.2	15.6	18.8		2	15.5	150
PFPE-3	8760	8760	8760			8760	9.5	20.5	12	28		18	180
ES-7	8760	8760	8760			8760	24.7	11	15		6	19.2	150
M-5d	8760	8760	8760			8760	21.5	22.8	17.3	20		20.4	150
M-11	8760	8760	8760			8760	24.3	21.1	20.6	17.	က	20.8	180
M-5	8760	8760	8760			8760	,23	31	12	25		22.8	180
M-13	8760	8760	8760			8760	37.8	34.6	26	19.		29.4	150
M-1	8760	8760	6790			8268	6.5	14	36.5	11		17	180
Si-3	5409	8760	8760			7922	32	12.5	12	14		17.5	180
M-12	8760	8760	4232			7628	14.7	14.6	46.4	15.	o c	22.9	150
	6261	6313	8760			7524	47.4	46.6	22.8		1	38.5	180
PFPE-5	8760	8760	8760			7274	S	3.5	1.5			∞	150
ES-1	5783	8760	5497			7200	44.5	16	27			33.5	180
M-2	8760	8760	1848			7032	27	56	46	20		30	180
Si-2	8760	1557	8760			6957	က	9.2	ა	S		5.5	150
FS-2	685	8760	8760			5972	15	19		25.	2	20.2	150
Si-5	5577	8760	629	8760		5932	40.3	5.1	31.8	œ	2	21.4	180
Δ.	4977	4737	5926	6586		5557	84		66.5			74.3	180
ES-3	3345	3501	2117	4340		3326	92	69.5	68.5	89		70.5	180
		Ho	urs to	Hours to Failure ^a				Wei	Weight Loss	q(%) s			Cvcle
						Aver-						Aver-	Time
Lubricant		2	က	4	5	age		2	3	4	5	age	(s)
PFPE-1	43800	43800 ^e	43800	11116	43800	37263	18.7	18.9	39.5	39.4	15.7	26.4	150
PFPE-2	43800	43800	43800	43800e	43800		19.6	8.5	12	15.9	13.5	13.9	150
M-3	43800	39210	24431e	19544e	20276e		27.6	29.7	20.4	24.7	22.8	25	150
2-10	10000	40114	COSCT	29133	390400	32383	1.8	20.02	23.5	13	24.5	19.9	150

а. О

. . .

Or to end of test (1 year = 8760 hr and 5 years = 43800 hr.

Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4 or motor Nos. 1 through 5).

Baked in vacuum at 100°C for 20 hr.

Royco 49B (Table 1).

Drive motor failed. Bearings not removed from armatures. Since the last status report, armatures with bearings assembled in new motors were further tested in Start-Stop tests.

TABLE 8. LUBRICANT RATING CHART

Lube Code		Oxi Envir b	dizing onment c	Vacuum (38°C)	Vacuum (93.3°C)	Vacuum Start-Stop	Low Temperature Start	Decision (See Note)
KG 80	M-1	19	8	20		1.		,
SRG 200	M-2	1 5	l °	8.5	6	14 20	23	1
Aeroshell 5	M-3	6	8	8.5	6,6 ^c	7	26	
Royco 24R	M-4	"	ľ	31	0,0	1 '	22	,,,
Royco 49	M-5	l		8.5	6	, ,	5	EL
Royco 49B	0	16	8	0.0	13 ^c	7 7	19	
Aeroshell 14	M-6] 10	ľ	35	13	′		
Aeroshell 16	M-7	1		24			6	EL
Apiezon L	M-8			27		1		EL
Unitemp	M-9		1 1	29	i			EL
Mobilgrease 28	M-10	6	8	8.5	27	17		EL
Conoco HD #2	M-11	6	19	17	18	7	20	
BP2110	M-12	12	8	8.5	14	16		
Exxon Andok C	M-13	6	16	8.5	23	7	14	
Supermil 06752	ES-1	6	1 10	23	43	,	18	
Aeroshell 17	ES-2	, u		32		19	8	EL
Aeroshell 7	ES-3]	34	32	0.5		EL
L-11G	ES-4			33	32	25	10	EL
Exxon 5182	ES-5	20	8	8.5	24		7	EL
Beacon 325	ES-6	15	8	28	28	7	12	
BP 8135	ES-7	18	8	8.5	31	,	15	
DC No. 33	SI-1	6	١	21	or .	7	13	
G-351	Si-2	17	8	8.5	16,20 ^C	21	01	EL
Supermil 31052	Si-3	-,	"	30	25	15	21	
G-330M	Si-4			8.5	19	19	1	EL
G-341L	Si-5	6	8	18	22	23	17	EL
3L27-2	Si-X	6		26	30	23	9	
FS-1281	FS-1	21		34	30			EL
FS-1290	FS-2	6	8	8.5	12	99		EL
Kel-F No. 90	FCC-1	•	١	0.0	29	22	l	i
803	PFPE-1	6	8	8.5	6,17 ^c	7	,,	EL
3L-38RP	PFPE-2	22	18	8.5	6,17c	7	11	j
3L-38RP Baked		25	**	8.5	6		3	j
631A	PFPE-3	6	8	8.5	6	7 7	4	į
240AZ	PFPE-4	13	8	22	26	24	25 16	
240AB	PFPE-5	14	20	25	6	18	10	
240AC	PFPE-6	24	8	8.5	6	7	94	
3L-38-MS	PFPE-7	23	17	19	21	7	24 2	j

Note: EL - eliminate from further testing.

a. Air, 90% RH.b. 10 psi O₂, 90% RH.

c. 10-15 percent fill, all others 25-30 percent fill.

to illustrate the comparative principle only; however, using this chart, it was previously decided to eliminate 15 of the materials from further testing because they performed poorly in either the vacuum ambient or vacuum high temperature tests.

Since the last status report, four more oxidizing environment and two startstop tests have been completed.

Due to the present and future emphasis on the Space Station program and since this is the last report in this series of status reports of the lubricants specified in Table 1, a new series of status reports is forthcoming on an updated group of lubricants which will be replacing those for the most part in the aforementioned table.

VI. CONCLUSIONS

Since testing has been completed in this program, the following conclusions, from the 1-year and 5-year vacuum tests data, have been made:

- 1) As a whole, the chemical class listed as PFPE in Table 1 has given the best results in all the vacuum tests completed.
- 2) In the 1-year vacuum ambient temperature tests, PFPE-2 (as manufactured and vacuum baked) and PFPE-6, Si-2 and Si-4, M-5 and M-12, and ES-5 have given the best results with less than a 10 percent average weight loss. In the 5-year vacuum ambient temperature tests, PFPE-2 and M-3 have given the best results with less than a 14 percent average weight loss.
- 3) In the 1-year vacuum high temperature tests, M-5 and all the PFPE greases, except PFPE-4 and PFPE-7, have given the best results with less than a 20 percent average weight loss. In the 5-year vacuum high temperature test, one PFPE-2 motor and one M-3 motor completed the test with weight losses of 15.7 and 24.5 percent, respectively.
- 4) In the 1-year start-stop tests, ES-5, M-3, and PFPE greases (except PFPE-3, PFPE-4, and PFPE-5) have given the best results with less than a 10 percent average weight loss. In the 5-year start-stop test, PFPE-2 has given the best results with a 13.9 percent average weight loss. Since there were seven motor failures in the test, further testing has been completed on the armatures with bearings (see note e of Table 7).
- 5) A 25 to 30 percent fill of a grease gives better results, on the whole, than a 10 to 15 percent fill.
- 6) The use of PFPE-2 as received gives better results, on the whole, than using PFPE-2 after a vacuum bake at 100°C for 20 hours.

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APPROVAL

AN EVALUATION OF GREASE TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS (Final Status Report No. 8)

By E. L. McMurtrey

The information in this report has been reviewed for security classification. Review of any information concerning Department of Defense or nuclear energy activities or programs has been made by the MSFC Security Classification Officer. This report, in its entirety, has been determined to be unclassified.

R. J. SCHWINGHAMER

Director, Materials and Processes Laboratory